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Design, Fabrication and Testing of 30-cm dia. Dished Carbon-Carbon Ion Engine Grids

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Two sets of 30-cm dia. dished carbon-carbon Ion engine grid sets were fabricated, assembled and tested. The carbon-carbon grid blanks were fabricated from unidirectional prepreg tape, having a very high fiber volume fraction of 57% and resulting in the strongest quasi-isotropic carbon-carbon material known to the authors. The flexural modulus of the panel material was measured at 275 MPa, corresponding to 80% of the value for molybdenum. Grid hole apertures were machined by laser-cutting. Inspection of the grid after fabrication revealed very tight hole tolerances, however, radii of curvature of the screen, accelerator and decelerator grids were larger than specified. The data indicate a correlation between radius of curvature and open-area-fraction of the grid, possibly caused by the laser-fabrication process. To minimize the impact of the larger screen grid radius of curvature relative to the accelerator grid, the grids were mounted dished inward. Pervance and electron backstreaming limits were measured. At 1.7 A screen current the pervance limit was found to be 1.4 kV and at 1.0 A screen current it was 1.0 kV. The electron backstreaming limit was -130 V at 1.7 A and -95 V at 1.0 A.

Introduction

Carbon-carbon grid technology for use in ion engine accelerator systems has been under investigation by various researchers¹⁻¹⁰ for several years now. Initial results of these investigations have been very encouraging due to material properties which make carbon-carbon attractive for use as a grid material. Carbon-carbon composites consist of a graphitic fiber in a carbonaceous matrix, consisting in turn partly of a graphitic and a glassy carbon phase. Due to this composition of materials, carbon-carbon exhibits the low sputter yields of graphite, yet at considerably higher structural strength than graphite alone. Blandino et al.¹¹ was able to show that carbon-carbon wear rates due to xenon ion sputter erosion are very similar to those found for graphite and consequently obtained wear rates of carbon-carbon vs. molybdenum of 1:7.3 and 1:7.7 at xenon ion energies of 500 eV and 750 eV, respectively.

This reduced wear rate translates directly into longer ion accelerator grid lifetimes which will increase overall ion engine life. Typical ion engine wear requirements are many thousand hours for missions to the

outer planets, the Kuiper-Belt, the main belt asteroids, and comets. The NASA Solar Electric Propulsion Technology Applications Readiness (NSTAR) program, which seeks to validate ion engine technology for the same types of missions, has a service life requirement of 8,000 hours, corresponding to a xenon throughput of 83 kg. Often mission scenarios for outer planet and small body missions involve the use of 1 ion thruster operating sequentially to achieve the required total impulse. The use of ion engines equipped with carbon-carbon grid technology may eliminate grid erosion as the life-limiting engine phenomena. Longer engine life will enable the use of fewer engines and reduce the propulsion system dry mass, the initial spacecraft wet mass and overall mission cost.

Previous investigations of carbon-carbon grid technology have focused on small (10 - 15 cm dia.) grids to demonstrate the feasibility of this technology, and were focused primarily on grid fabrication techniques, measurement of material properties such as coefficient of thermal expansion (CTE) and sputter yield, and grid performance parameters (pervance and electron backstreaming). Most recently, two 700-hr ion engine endurance tests with 15-cm grid sets were completed at JPL¹, showing no detectable evidence of carbon-carbon grid erosion over the duration of the tests. In addition, a unique grid configuration with non-circular grid hole apertures has been investigated².

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This paper focuses on the development of 30-cm carbon-carbon grid optics retrofitable to the NSTAR ion engine. The goal of this activity is to investigate the feasibility of fabrication, assembly and operation of large diameter carbon-carbon grids as a follow-on to the current NSTAR technology. Unique design issues to the fabrication, assembly and test of large diameter carbon-carbon grids were to be identified and lessons-learned from this activity will be applied to future designs,

Grid Set Design, Fabrication and Inspection

Grid Panel Fabrication

The carbon-carbon panels for the 30-cm dia. grids were fabricated in part at JPL, B.F. Goodrich Aerospace, and Accu-Tech Laser Processing, Inc. Grid design, lay-up and curing of the panels were performed at JPL. Chemical vapor infiltration (CVI), carbonization and graphitization stages of the grid fabrication were performed at B.F. Goodrich Aerospace and machining of the grid apertures, mounting holes and final machining of the grids were executed at Accu-Tech Laser Processing, using a laser-cutting technique.

The first of the fabrication phases consisted of laying up of the prepreg material in the desired orientation. Here, the lay-up was of the $[0^\circ/+60^\circ/-60^\circ]_s$ type and was performed at JPL using commercial unidirectional prepreg material. Each grid panel consists of a total of six plies, with fibers aligned along -60° , $+60^\circ$, 0° , 0° , $+60^\circ$ and -60° directions in the six different plies, respectively. The lay-up is symmetrical with respect to the centerplane to prevent warpage due to the highly directional thermal characteristics of the plies^{1,3}. Commercial prepreg tape from YLA Inc., consisting of DuPont E55 fiber made from pitch based carbon and phenolic resin (Borden SC1008 phenolic) was used for the panel lay-up. A total of six 30-cm grid panels were fabricated, providing enough material for the fabrication of 3 two-grid sets or 2 three-grid sets.

Fabrication of the grid panels is discussed in detail in Ref. 1. Table 1 shows some of the material properties of the carbon-carbon when compared with carbon-carbon panels of a previous production run for the fabrication of 15-cm dia. grid panels¹. Material properties of the new grid material were obtained by using flat panel material samples from a fabrication run for 15-cm dia. grid panels (described in detail in Ref. 1), since the dished carbon-carbon panels were too expensive to be sacrificed

for destructive material testing. However, the dished carbon-carbon material is identical to the flat carbon-carbon panels of Ref. 1 since it was fabricated from the same prepreg material, featured the same $[0^\circ/+60^\circ/-60^\circ]$ lay-up and underwent the same fabrication procedure.

The high fiber volume and corresponding low carbonized resin content in the new $[0^\circ/+60^\circ/-60^\circ]_s$ panels, as indicated in Table 1, is significant in that it resulted in the strongest quasi-isotropic carbon-carbon material known to the authors. The marked increase in the flexural modulus over the previous generation of panels is of particular importance for the intended use of these panels as ion engine grids. The flexural modulus is a critical grid structural property since it determines how far the grid may bend due to electrostatic or Vibrational (launch) loads. The value for the flexural modulus in the 0° direction of the carbon-carbon panels now reaches about 80% of the value for molybdenum. However, this increase in flexural modulus in the 0° direction comes at a price. Due to the higher fiber volume of these panels, less cross-linkage is provided by the reduced matrix content and the flexural modulus in the 90° direction is thus reduced over the value found for the previous generation of panels. Thicknesses for the dished panels were measured at 0.46 mm for all types, i.e. screen accelerator and decelerator grids. Screen grid panels, which have a somewhat smaller diameter, weigh around 95 grams whereas accelerator and decelerator panels weigh around 108 grams and 105 grams, respectively.

The dished grid blanks were then shipped to Accu-Tech Laser Processing for laser-machining using a 500" W slow flow coaxial CO_2 laser by Photon Source CCs at a 150 W power setting. Spot size of the laser was 0.005 inch (0.13 mm). Compressed air was used as a cutting gas to remove debris from the drill site¹. No reactive gases were used to aide in the cutting. The grid holes were machined by orbiting the narrowly focused laser beam along a path that was to become the hole edge. Earlier tests conducted by the authors with other vendors, using the laser as a drill rather than a cutting lead to disappointing results⁴. However, using the cutting technique mentioned above resulted in exceptional hole quality and outstanding accuracy and repeatability in hole diameter. This laser-cutting technique is very similar to the one explored previously by Meserole and Hedges^{7,9}.

The laser machining included the cutting of the grid holes (about 15,500 per grid), as well as the grid mounting holes and finish cutting the grid diameters. Screen, accelerator and decelerator grid dimensions were

Table 1: Material Properties of Carbon-Carbon Panels

| Property | Previous Fabrication [0°/+60°/-60°] _s | Current Fabrication [0°/+60°/-60°] _s |
|------------------------------|---|--|
| Density (g/cm ³) | 1.77 | 1.83 |
| Thickness (mm) | 0.5(1)/0.95(2) | 0.46 |
| Fiber by Weight | 50.2% | 69% |
| Resin by Weight | 17.6% | 3-4% |
| CVI by Weight | 32.2% | 26-28% |
| Fiber Volume | 40.1% | 57% |
| Resin Volume | 16.4% | 3% |
| CVI Volume | 30.0% | 2-4% |
| Void Volume | 13.6% | 13% |
| Tensile Strength (MPa) | 308 | 486(3) |
| Tensile Modulus (GPa) | 179 | 145(3) |
| Flexural Strength (MPa) | 240(1)/280(2) | 205(3)/117(4) |
| Flexural Modulus (GPa) | 210(1)/161(2) | 275(3)/61(4) |

(1)Screen Grid

(2)Accelerator/Decelerator Grid

(3)0° Direction

(4)90° Direction

dictated by the requirement to make this grid set retrofittable to the NSTAR engine. Photographs of the three of the six grids are shown in Figs 1, 2, and 3, respectively. Two tags are visible on all three grids shown in these figures. The arrow indicates the, clocking hole with respect to which contour and grid hole measurements were taken. The second tag is a grid identification mark.

Grid dimensions for the three grid types are listed in Table 2. The screen grid has an outer diameter of 35.06 cm with the hole pattern area extending over a diameter of 28.58 cm (active beam area). The dished area of the screen grid is 33.17 cm in diameter, leaving a flat annulus around the dish which serves as a mounting flange for the grid. The design value for the radius of curvature for the grid was 203.2 cm, however, as will be discussed below, the final dish depth was found to be smaller than the design value. Twelve mounting holes were laser-cut equidistantly on a bolt circle of 34.11 cm into the flat flange outside the dished region. The grid hole sizes were measured to be

1.89 mm dia. with very little deviation (+ 0.0025 mm/- 0.009 mm). Grid hole spacing is 2.21 mm center-to-center on all three grids. The open area fraction of the grid can then be calculated according to

$$OAF = \frac{\pi d^2}{6c^2 \tan 30^\circ} \quad (1)$$

where d is the hole diameter and c the center-to-center hole spacing. For the values given above, the screen grid open area fraction is 66%.

The accelerator grid is of the SIIAG-type (small hole accelerator grid) and larger in diameter (38.06 cm) than the screen grid. Twelve mounting holes are located on a 36.73 cm dia. bolt circle. Clearing holes for screws mounting the decelerator grid are located on the same bolt circle, clocked 15° and clearing holes for the heads of screws used to mount the screen grid beneath the

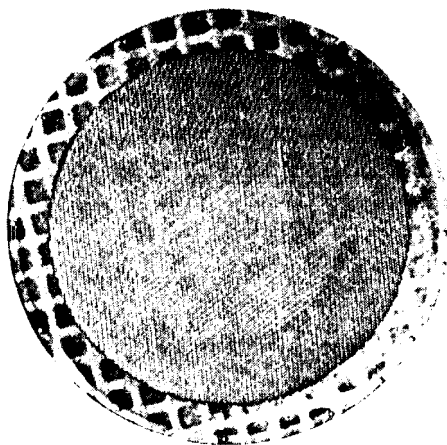


Fig. 1: Screen Grid

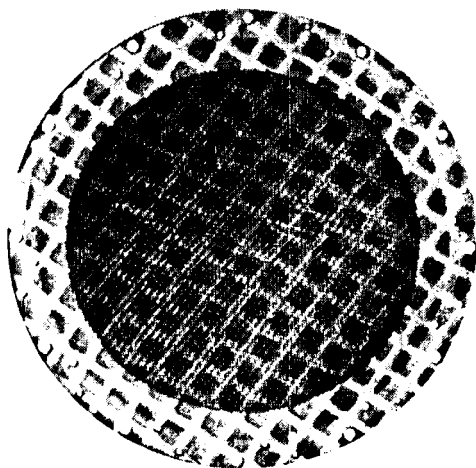


Fig. 2: Accelerator Grid

accelerator grid are located on a 34.11 cm bolt circle. Clearing holes for the screw heads prevent contact between grids. The accelerator grid dished area is the same as the screen grid. Hole diameters were measured to be 1.13 mm and again very little deviation from the mean hole diameter can be noted (± 0.02 mm). From the average hole diameter and the grid hole spacing of 2.21 mm, the open area fraction of the accelerator grid is calculated to 23.5 %.

The decelerator grid has the same OD as the accelerator grid and the same dished area as the screen and

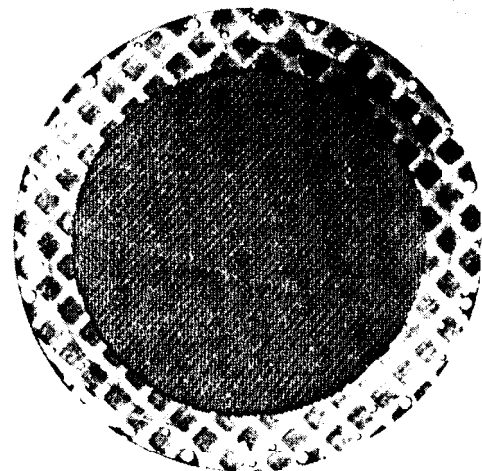


Fig. 3: Decelerator Grid

accelerator grid. The bolt circles for the grid mounting holes are identical to the ones on the accelerator grid but clocked 1 SO. The decelerator grids include clearing holes for screen grid and accelerator grid mounting screw heads. Decelerator hole diameters are 1.77 mm on average, resulting in an open area fraction for the decelerator grid of 58 %.

Mount Ring Fabrication

In order to save weight for the total assembly and match CTE's with the grids, a carbon-carbon mount ring was fabricated for the 30-cm grid set. A photograph of the mount ring is shown in Fig. 4. The mount ring was laid up from segments of plain weave fabric made from PAN based Amoco T-300 fiber and phenolic resin (Borden SC1008). Segments of the plain weave fabric are still visible in the finished product, shown in a closer view in Fig. 5. In this figure, the mount ring is shown before machining of the mounting holes. The lay-up and curing were performed with the help of an aluminum fixture shown in Figs. 6 and 7. The mount ring was laid up in the U-shaped crevice formed by the fixture, giving the ring its shape. After completion of the lay-up, a long slab of silicon rubber was inserted into the crevice, covering the composite lay-up and fixated with a large hose clamp. The entire assembly was then placed in a circulated air oven at 175 °C for 16 hours for curing. Pressure was provided by thermal expansion of the silicon mandrel.

Table 2: Description of 30-cm Carbon-Carbon Grids

| Description of 30-cm Carbon-Carbon Grids | |
|--|----------------------|
| Common to all Grids | |
| Grid Material | Carbon-Carbon |
| Grid Shape | Dished |
| Beam Diameter (cm) | 28.6 |
| Hole Quantity | -15,500 |
| Hole Spacing (mm) | 2.21 |
| Screen Grid | |
| OD (cm) | 35.(K) |
| OD Dish (cm) | 33.17 |
| Thickness (mm) | 0.46 |
| Hole Dia. (mm) | 1.89 (+0.025/-0.009) |
| Open Area Fraction | 66 % |
| Weight (gram) | 43.6 |
| Manufacturing Technique | Laser Drilling |
| Accelerator Grid | |
| OD (cm) | 38.06 |
| OD Dish (cm) | 33.17 |
| Thickness (mm) | 0.46 |
| Hole Dia. (mm) | 1.126 (+/- 0.02) |
| Open Area Fraction | 23.5 % |
| Weight (gram) | 80.2 |
| Manufacturing Technique | Laser Drilling |
| Decelerator Grid | |
| OD (cm) | 38.06 |
| OD Dish (cm) | 33.17 |
| Thickness (mm) | 0.46 |
| Hole Dia. (mm) | 1.77(+0.008/-0.01) |
| Open Area Fraction | 58 % |
| Weight (gram) | 58.4/60.6" |
| Manufacturing Technique | laser Drilling |
| Mount Ring | |
| ID (cm) | 33.17 |
| OD Upper Flange (cm) | 35.06 |
| OD Lower Flange (cm) | 38.06 |
| Material | Carbon-Carbon |
| Weight | 222.6 |
| Manufacturing Technique | Mechanical |

"Two grids weighed

Final machining of the mount ring was performed by Electro-Tech Machining. The mount ring has an ID of 33.17 cm, equal to the diameter of the dished area of the grids and two flanges. The screen grid is mounted to the upper flange, shown in Fig. 4. Ceramic spacers are mounted to the lower flange, which in turn support the accelerator and decelerator grids. The OD of

the upper flange is identical to the OD of the screen grid (i.e. 35.06 cm) and the OD of the lower flange is identical to the ODs of the accelerator and decelerator grids (i.e. 38.06 cm). The lower flange, also features six mounting holes for attaching the mount ring to the ion engine. These mounting holes are placed uniformly spaced on a 36.19 cm dia. bolt circle. Overall height of the mount

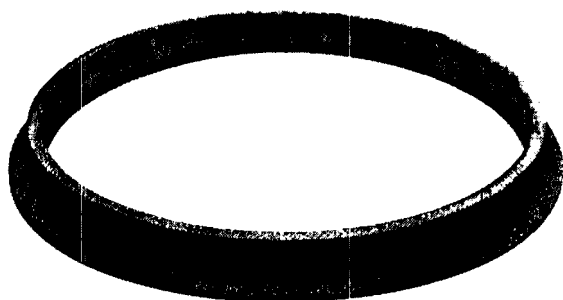


Fig. 4: Carbon-carbon Mount Ring

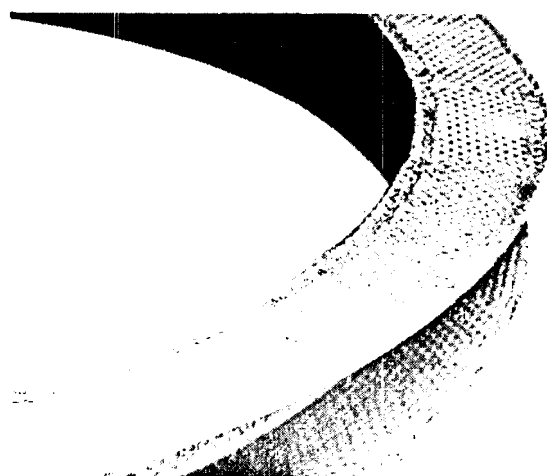


Fig. 5: Close-up of Carbon-Carbon Mount Ring showing Details of Lay-up



Fig. 6: Aluminum Fixture used in Mount Ring Fabrication (Assembled)



Fig. 7: Aluminum Fixture for the Mount Ring (Disassembled)

ring, measured from the bottom surface of the lower flange to the top surface of the upper flange, is 3.48 cm (+0.1/-0.3 cm).

Inspection of the mount ring after the carbonization, graphitization and CVI phases of the fabrication process revealed that the two flanges (upper and lower) of the ring were not parallel, probably as a result of the high temperature fabrication processes and the fact that the mount ring was high-temperature treated free-standing, because the aluminum and silicon rubber tooling was not able to withstand the temperatures. The upper flange drops with respect to the lower flange by about 0.2 - 0.6 mm toward the outer edge of the upper flange depending on location along the circumference.

Figure 8 shows an assembled grid set consisting of the mount ring and two grids, a screen and accelerator grid. The overall grid set mass is 550 grams. Although difficult to recognize in Fig. 8, the grids are mounted dished inwardly. The reason for mounting the grids in this fashion was to minimize the performance impact of the difference in screen and accelerator grid dish depths. The screen grid has a slightly larger radius of curvature than the accelerator grid. Mounting the grids dished outward from the engine would have resulted in a larger grid gap between the screen and accelerator grids near the center of the grids and a smaller gap towards the edges. The twin current density distribution of an ion engine generally peaks on the centerline. By dishing the grid set inward, a smaller grid gap could be maintained near the center of the grid, leading to a higher extractable beam current.

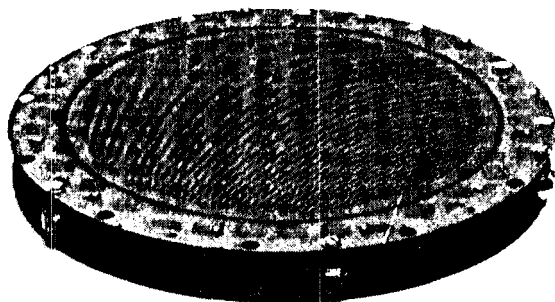


Fig. 8: Assembled Two-Grid 30-cm dia. Carbon-Carbon Grid SC

Grid-Set Inspection

Following the fabrication of the grids, measurements of grid hole diameters, the radius of curvature, and the grid gap between the screen and accelerator grid of the mounted on the carbon-carbon mount ring were performed.

Measurement of Grid Hole Diameters

The measured grid hole sizes for the different grids are given in Table 2. Grid hole measurements were performed with a MicroXcel optical measurement system equipped with a Ram Optical Instrumentation camera. Measurements using this system are conducted by focusing a camera with a fixed focal length on one hole edge. A highly magnified image (up to 250x magnification) is transmitted for display on a monitor which allows the operator to adjust the focus of the image by moving the camera platform accordingly. The position of the camera platform in x-, y-, and z-coordinates is determined. The camera is then moved until the opposite hole side is brought into focus and the system provides an x, y-, and z-read out for this new position. Noting the distance the camera has traveled in x-, y-, and z-directions allows the determination of the hole diameter.

Inspection of Grid Curvature

Measurements of the radius of curvature of the grids were performed with the same optical technique used to determine the grid hole diameter described above. Here, the grid was placed onto a flat bench (equal or less than

0.008 mm/m variation), serving as the horizontal reference plane, with the convex side of the grid facing up. The fixed focal-length camera was first focused onto the top (convex) surface of the grid by positioning it properly with respect to the grid surface. Then, by moving it downward in the +z direction (compare with Fig. 9), the camera was focused onto the flat bench surface beneath the grid. Noting the shift in the z-coordinate of the camera platform allowed the determination of the height of the top surface above the surface of the flat bench. The x- and y-coordinates were also noted to determine a height profile of the grid. An (x, y, z) coordinate system was defined with respect to the grid as shown in Fig. 9. The origin was placed at the center (apex) of the grid and the +x direction pointed toward the clocking hole identified by the arrow-mark in Figs. 4 through 8. The +z-axis faced downward toward the bench.

The measured height of the upper grid surface above the bench surface (labeled z in Tables 3 through 5 below) was then compared with the calculated value for this height (labeled z_{calc} in the tables below) based on the design value for the radius of curvature. For a spherical dish, this value is given by

$$z_{calc} = R \cdot \sin(\cos^{-1} \frac{\sqrt{x^2 + y^2}}{R}) \cdot \sqrt{R^2 - s^2} \quad (2)$$

where R is the radius of curvature for the top surface of the grid, x and y are the coordinates of the grid location under consideration, with the origin located at the grid, and s is the radius of the dished area as shown in Fig. 10. Note that the design value for the radius of curvature was given for the lower grid surface, while the height z of the grid surface is measured to the top surface of the grid,

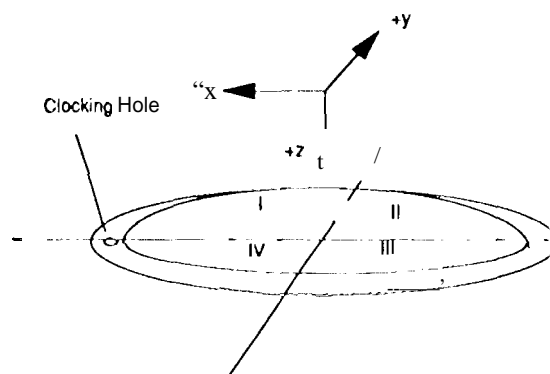


Fig. 9: Coordinate System for Grid Contour Measurements

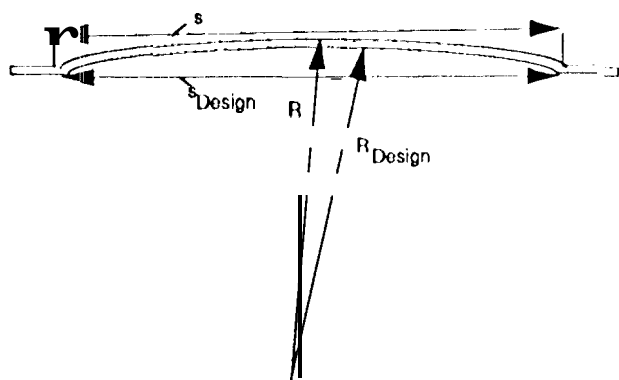


Fig. 1(): Geometry of the Dished Grid

For this reason, the value for R in Eqn (1) has to be increased by the grid thickness, i.e.

$$R = R_{Design} + t \quad (3)$$

where R_{Design} is the specified design value for the radius of curvature as measured at the lower grid surface, i.e. 2,032 mm and t is the grid thickness of 0.46 mm. Note further that due to measuring the grid height to the upper grid surface, ideally the value for the radius of the dish, s , also has to be increased according to

$$s = s_{Design} \left(1 + \frac{t}{R_{Design}}\right) \quad (4)$$

where s_{Design} is the specified design value for s , i.e. 168.9 mm and t is the grid thickness again. The value in brackets is very close to unity and subsequently the value for s was approximated by the design values s_{Design} given for the lower surface.

Using the calculated value for the upper grid surface height over the horizontal reference plane (i.e. the bench), z_{cal} , the difference of the measured height from the design height can be determined and the deviation from the ideal grid shape can be quantified. Table 3 lists the corresponding values for one of the two screen grids that were fabricated. From Table 3, it is clearly evident that the screen grid surface is significantly less curved than the design intended. The differences between actual height and the design value of the height vary across the grid, decreasing toward the edges of the grid. However, close to the center of the grid, the apex of the grid surface is about 1 mm lower than its design value. Even though the grid does not appear to be perfectly spherical, an approximate

radius of curvature of the screen grid based on the curvature measurements may be determined from

$$R_{actual} = \frac{z_{max}^2 + s^2}{2 z_{max}} - d \quad (5)$$

where d is the grid thickness (0.46 mm). For a maximum height of about 5.7 mm, the approximate radius of curvature is 23.7 cm, or about 3.5 cm larger than its design value of 20.3 cm.

Inspecting Table 4, the accelerator grid also shows a flattened profile. When compared with its design contour, however, the differences are less pronounced than for the screen grid. Here, the actual grid apex height falls short by about 0.3 mm from its design value. Also, slight non-symmetries in the profile of the accelerator grid can be noted when inspecting grid heights measured along the horizontal and the diagonal intersecting the second and fourth quadrant of the grid (compare with Fig. 9). These non-symmetries may be due to grid warping and at opposite sides of the grids at equal distance from the apex these differences amount to about 0.3 - 0.4 mm. Even though a balanced [0°/±60°/-60°] lay-up was chosen to limit grid warpage, slight imbalances in the lay-up may have occurred due to small, localized differences in the thickness of the prepreg material caused by splits in the tape for example.

Based on the measured apex height of approximately 6.45 mm, the radius of curvature may be estimated using Eqn. (5) with the assumption that the grid profile is spherical. The radius of curvature of the accelerator grid is then determined to 21.35 cm, almost 2 cm smaller than the value calculated for the screen grid. This difference in radius of curvature for the two grids is the source of the difficulties encountered when trying to mount them with an outward dish. The shallower screen grid mounted beneath the more sharply curved accelerator grid will leave a bigger gap between the two grids near the center (see Fig. 11). By mounting the grids with the convex side of the dish facing toward the engine, the sharper curved accelerator grid mounted above, the screen grid leaves a smaller grid gap near the grid center. It should be noted that changing the grid orientation in this fashion was only possible because placement of the grid holes was not compensated for the grid curvature and holes were cut parallel to the axis of rotational symmetry for the grid rather than normal to the grid surface.

Table 3: Contour Measurements of Screen Grid

| x | y | z | z calc | z-z calc | delta z |
|----------------|---------|------|--------|----------|---------|
| (mm) | (mm) | (mm) | (mm) | (mm) | (%) |
| Vertical | | | | | |
| 125.94 | -2.02 | 2.64 | 2.87 | -0.23 | -8.01 |
| 98.71 | -2.29 | 3.81 | 4.38 | -0.57 | -12.92 |
| 73.06 | -3.41 | 4.70 | 5.46 | -0.76 | -13.96 |
| 31.42 | -4.93 | 5.63 | 6.53 | -0.90 | -13.79 |
| 6.29 | -9.39 | 5.75 | 6.75 | -1.00 | -14.80 |
| 5.69 | 2.25 | 5.78 | 6.77 | -0.99 | -14.56 |
| 5.32 | -0.09 | 5.79 | 6.77 | -0.98 | -14.55 |
| 4.06 | 6.57 | 5.72 | 6.76 | -1.05 | -15.50 |
| -24.14 | 5.54 | 5.69 | 6.63 | -0.94 | -14.18 |
| -48.90 | 4.63 | 5.16 | 6.18 | -1.02 | -16.50 |
| -105.08 | 2.57 | 3.37 | 4.06 | -0.69 | -17.06 |
| Horizontal | | | | | |
| 9.56 | -127.46 | 2.53 | 2.75 | -0.22 | -7.97 |
| 9.09 | -96.67 | 3.85 | 4.46 | -0.61 | -13.67 |
| 8.84 | -65.80 | 4.88 | 5.69 | -0.81 | -14.25 |
| 8.42 | -44.45 | 5.46 | 6.27 | -0.82 | -13.00 |
| 7.07 | -19.55 | 5.77 | 6.67 | -0.90 | -13.54 |
| 5.69 | 17.77 | 5.68 | 6.69 | -1.01 | -15.06 |
| 4.91 | 39.13 | 5.32 | 6.40 | -1.08 | -16.83 |
| 3.80 | 63.55 | 4.74 | 5.78 | -1.04 | -18.05 |
| 2.84 | 89.60 | 3.96 | 4.80 | -0.84 | -17.55 |
| 1.63 | 122.63 | 2.81 | 3.07 | -0.26 | -8.54 |
| Diagonal I-III | | | | | |
| 97.87 | 91.99 | 2.26 | 2.33 | -0.07 | -3.18 |
| 66.24 | 62.15 | 3.93 | 4.75 | -0.81 | -17.13 |
| 39.39 | 37.81 | 5.02 | 6.04 | -1.02 | -16.88 |
| 20.35 | 23.09 | 5.51 | 6.54 | -1.03 | -15.82 |
| -9.11 | -16.05 | 5.78 | 6.69 | -0.92 | -13.72 |
| -28.91 | -29.84 | 5.50 | 6.35 | -0.86 | -13.48 |
| -51.01 | -52.07 | 4.66 | 5.47 | -0.81 | -14.89 |
| -73.55 | -65.10 | 3.76 | 4.40 | -0.64 | -14.63 |
| -91.86 | -82.15 | 2.72 | 3.04 | -0.32 | -10.47 |
| -100.10 | -96.39 | 1.99 | 2.02 | -0.04 | -1.75 |
| Diagonal II-IV | | | | | |
| -90.07 | 88.38 | 2.69 | 2.86 | -0.16 | -5.67 |
| -62.63 | 58.89 | 4.25 | 4.96 | -0.71 | -14.31 |
| -43.53 | 37.20 | 5.12 | 5.97 | -0.85 | -14.25 |

Table 4: Contour Measurements of Accelerator Grid

| x | y | z | z calc | z-z calc | Delta z |
|----------------|---------|------|--------|----------|---------|
| (mm) | (mm) | (mm) | (mm) | (mm) | (%) |
| Vertical | | | | | |
| 137.21 | 2.02 | 2.09 | 2.14 | -0.05 | -2.45 |
| 112.00 | 0.73 | 3.60 | 3.69 | -0.09 | -2.52 |
| 84.24 | 1.41 | 4.84 | 5.03 | -0.19 | -3.88 |
| 48.43 | 1.21 | 5.94 | 6.20 | -0.26 | -4.14 |
| 21.15 | 0.87 | 6.41 | 6.67 | -0.26 | -3.89 |
| 12.13 | 0.44 | 6.40 | 6.74 | -0.34 | -5.02 |
| 0.10 | 1.70 | 6.45 | 6.78 | -0.32 | -4.77 |
| -4.68 | 2.19 | 6.48 | 6.77 | -0.29 | -4.35 |
| -24.34 | 2.39 | 6.26 | 6.63 | -0.37 | -5.54 |
| -44.17 | 1.74 | 6.01 | 6.30 | -0.29 | -4.61 |
| -59.47 | 2.10 | 5.66 | 5.91 | -0.24 | -4.11 |
| -82.91 | 3.01 | 4.84 | 5.08 | -0.2s | -4.83 |
| -102.52 | 4.58 | 4.04 | 4.19 | -0.14 | -3.39 |
| -137.24 | 1.07 | 2.07 | 2.14 | -0.07 | -3.44 |
| Horizontal | | | | | |
| 0.85 | -131.01 | 2.60 | 2.55 | 0.0s | 1.86 |
| 1.01 | -69.39 | 5.38 | 5.59 | -0.22 | -3.86 |
| 2.37 | -55.15 | 5.79 | 6.03 | -0.24 | -3.97 |
| 3.11 | -38.22 | 6.15 | 6.42 | -0.27 | -4.20 |
| 1.13 | -14.89 | 6.44 | 6.72 | -0.28 | -4.1s |
| 1.98 | -6.63 | 6.44 | 6.77 | -0.32 | -4.76 |
| 1.13 | 3.18 | 6.45 | 6.78 | -0.32 | -4.74 |
| 0.34 | 10.48 | 6.44 | 6.75 | -0.31 | -4.62 |
| 0.15 | 16.78 | 6.38 | 6.71 | -0.33 | -4.89 |
| 1.58 | 35.25 | 6.18 | 6.47 | -0.29 | -4.51 |
| 1.19 | 58.13 | 5.64 | 5.95 | -0.31 | -5.17 |
| 1.04 | 79.65 | 4.91 | 5.22 | -0.30 | -5.83 |
| 0.33 | 100.93 | 3.98 | 4.27 | -0.29 | -6.73 |
| 1.20 | 118.51 | 3.05 | 3.32 | -0.27 | -8.10 |
| 1.31 | 133.14 | 2.18 | 2.41 | -0.23 | -9.43 |
| Diagonal I-III | | | | | |
| 83.37 | 76.6S | 3.39 | 3.62 | -0.23 | -6.41 |
| 71.22 | 63.62 | 4.28 | 4.53 | -0.26 | -5.64 |
| 57.47 | 52.01 | 5.03 | 5.30 | -0.27 | -5.05 |
| 45.50 | 38.67 | 5.57 | 5.90 | -0.33 | -5.55 |
| 31.61 | 25.73 | 6.05 | 6.37 | -0.32 | -4.97 |
| 26.70 | 20.76 | 6.18 | 6.50 | -0.31 | -4.80 |
| 18.01 | 12.67 | 6.32 | 6.66 | -0.34 | -5.09 |
| 11.83 | 6.76 | 6.38 | 6.73 | -0.35 | -5.26 |
| 4.00 | -1.12 | 6.47 | 6.77 | -0.30 | -4.49 |
| -7.26 | -12.42 | 6.44 | 6.73 | -0.29 | -4.25 |
| -19.78 | -21.83 | 6.31 | 6.56 | -0.25 | -3.85 |
| -28.08 | -34.07 | 6.05 | 6.30 | -0.25 | -3.98 |
| -47.48 | -46.87 | 5.44 | 5.68 | -0.24 | -4.31 |
| -54.30 | -58.3S | 4.99 | 5.21 | -0.22 | -4.23 |
| -66.37 | -67.93 | 4.43 | 4.56 | -0.13 | -2.86 |
| -79.90 | -79.57 | 3.60 | 3.65 | -0.05 | -1.40 |
| -88.68 | -90.19 | 2.79 | 2.84 | -0.05 | -1.74 |
| -97.87 | -98.88 | 1.99 | 2.01 | -0.02 | -0.82 |
| Diagonal II-IV | | | | | |
| -95.16 | 93.60 | 2.23 | 2.39 | -0.16 | -6.81 |
| -86.30 | 85.65 | 2.92 | 3.14 | -0.22 | -7.08 |
| -75.29 | 74.08 | 3.79 | 4.03 | -0.24 | -6.07 |
| -61.78 | 62.73 | 4.55 | 4.87 | -0.32 | -6.59 |
| -48.98 | 47.28 | 5.36 | 5.64 | -0.28 | -4.93 |
| -37.64 | 36.78 | 5.77 | 6.10 | -0.33 | -5.43 |
| -19.58 | 24.10 | 6.18 | 6.54 | -0.36 | -5.48 |
| -10.82 | 13.82 | 6.33 | 6.70 | -0.38 | -5.60 |
| 2.74 | -1.06 | 6.50 | 6.78 | -0.27 | -4.04 |
| 11.24 | -10.35 | 6.39 | 6.72 | -0.33 | -4.87 |
| 24.49 | -25.7S | 6.20 | 6.47 | -0.26 | -4.09 |
| 36.52 | -37.95 | 5.86 | 6.10 | -0.23 | -3.83 |
| 50.64 | -51.96 | 5.32 | 5.48 | -0.16 | -2.98 |
| 61.88 | -62.65 | 4.71 | 4.87 | -0.16 | -3.35 |
| 75.31 | -74.22 | 3.89 | 4.03 | -0.13 | -3.33 |
| 94.92 | -89.46 | 2.54 | — | — | 1.86 |

Table 5: Contour Measurements of Decelerator Grid

| x | y | z | z calc | z-z calc | Delta z |
|----------------|---------|------|--------|----------|---------|
| (mm) | (mm) | (mm) | (mm) | (mm) | (%) |
| Horizontal | | | | | |
| 136.09 | 4.50 | 1.81 | 2.21 | -0.40 | -18.00 |
| 74.51 | 0.24 | 4.74 | 5.41 | -0.67 | -12.32 |
| 53.99 | 0.64 | 5.44 | 6.06 | -0.63 | -10.31 |
| 42.37 | -0.04 | 5.71 | 6.34 | -0.62 | -9.84 |
| 19.19 | -0.90 | 6.09 | 6.69 | -0.60 | -8.92 |
| 4.72 | 0.10 | 6.18 | 6.77 | -0.60 | -8.79 |
| 3.39 | -0.21 | 6.19 | 6.78 | -0.59 | -8.64 |
| -4.45 | -0.05 | 6.20 | 6.77 | -0.58 | -8.53 |
| -29.15 | 1.06 | 5.97 | 6.57 | -0.60 | -9.13 |
| -44.45 | 0.73 | 5.74 | 6.29 | -0.56 | -8.84 |
| -65.37 | 0.50 | 5.19 | 5.73 | -0.53 | -9.29 |
| -94.69 | 0.82 | 4.07 | 4.57 | -0.50 | -11.04 |
| -117.77 | 0.39 | 2.99 | 3.36 | -0.38 | -11.17 |
| -135.61 | 1.26 | 1.96 | 2.25 | -0.29 | -12.90 |
| Vertical | | | | | |
| 1.60 | -134.72 | 2.09 | 2.31 | -0.22 | -9.41 |
| 0.30 | -106.55 | 3.48 | 3.98 | -0.50 | -12.51 |
| -0.85 | -80.61 | 4.64 | 5.18 | -0.54 | -10.39 |
| -0.94 | -58.77 | 5.37 | 5.93 | -0.56 | -9.46 |
| -0.58 | -40.28 | 5.75 | 6.38 | -0.63 | -9.93 |
| -0.48 | -23.09 | 6.06 | 6.65 | -0.59 | -8.82 |
| 0.75 | -0.94 | 6.17 | 6.78 | -0.60 | -8.90 |
| 0.73 | 7.37 | 6.21 | 6.76 | -0.56 | -8.27 |
| 1.18 | 30.16 | 5.95 | 6.55 | -0.60 | -9.19 |
| 1.87 | 49.91 | 5.60 | 6.16 | -0.56 | -9.14 |
| 2.76 | 82.88 | 4.48 | 5.09 | -0.61 | -11.95 |
| 3.28 | 109.30 | 3.30 | 3.83 | -0.53 | -13.94 |
| 6.35 | 134.66 | 1.95 | 2.30 | -0.36 | -15.49 |
| Diagonal I-III | | | | | |
| 89.61 | 99.71 | 2.05 | 2.35 | -0.30 | -12.94 |
| 69.74 | 77.11 | 3.55 | 4.12 | -0.57 | -13.87 |
| 45.83 | 55.89 | 4.91 | 5.49 | -0.58 | -10.52 |
| 26.68 | 39. *O | 5.64 | 6.22 | -0.59 | -9.45 |
| 9.42 | 22.07 | 6.07 | 6.64 | -0.56 | -8.49 |
| -10.74 | -6.31 | 6.18 | 6.74 | -0.56 | -8.31 |
| -29.04 | -22.78 | 5.88 | 6.44 | -0.57 | -8.81 |
| -50.16 | -37.39 | 5.28 | 5.81 | -0.54 | -9.27 |
| -62.61 | -50.16 | 4.62 | 5.19 | -0.57 | -11.05 |
| -75.35 | -63.71 | 3.86 | 4.38 | -0.52 | -11.82 |
| -88.75 | -80.25 | 2.87 | 3.25 | -0.39 | -11.84 |
| -96.84 | -92.78 | 2.10 | 2.35 | -0.25 | -10.45 |
| Diagonal II-IV | | | | | |
| -105.98 | 82.66 | 2.06 | 2.33 | -0.27 | -11.68 |
| -94.27 | 75.26 | 2.84 | 3.20 | -0.36 | -11.12 |
| -79.66 | 57.90 | 3.89 | 4.39 | -0.50 | -11.38 |
| -59.40 | 47.13 | 4.84 | 5.36 | -0.52 | -9.73 |
| -39.46 | 31.61 | 5.50 | 6.15 | -0.65 | -10.61 |
| -25.50 | 19.44 | 5.89 | 6.53 | -0.64 | -9.77 |
| -10.60 | 7.39 | 6.14 | 6.74 | -0.59 | -8.80 |
| 21.87 | -25.03 | 5.96 | 6.51 | -0.54 | -8.34 |
| 33.24 | -30.05 | 5.65 | 6.28 | -0.64 | -10.15 |
| 54.04 | -63.23 | 4.48 | 5.08 | -0.59 | -11.72 |
| 72.11 | -67.67 | 3.79 | 4.37 | -0.58 | -13.24 |
| 103.44 | -87.99 | 1.91 | 2.24 | 0.33 | 14.69 |

The data in Table 5 indicate that the decelerator grid contour has flattened to a degree ranging between the radii of curvature of the screen and accelerator grid. The apex of the grid is about 6.2 mm, or about 0.6 mm lower than its design value. The equivalent radius of curvature according to Eqn. (5) is 22.21 cm. Again, some non-symmetries can be noted in the decelerator grid height distribution, in particular along the horizontal axis listed in Table 5. However, differences at opposite sides of the grid along this horizontal axis differs by only 0.1 mm and may again be due to a slight warpage of the grid.

The height difference between the actual grid height and its design value at the center of the grid correlates well with the open area fraction of the grid (Fig. 12). The larger the open area fraction of the grid, the larger is the shortfall in grid height at the center from its design value. Although not yet fully understood, it is possible that residual stresses that remained in the grid panel after fabrication may be responsible for this behavior. Since for the grids with the larger open area fraction less grid material remains to counter these residual stresses, they shallow to a higher degree than the grids with lower open area fractions.

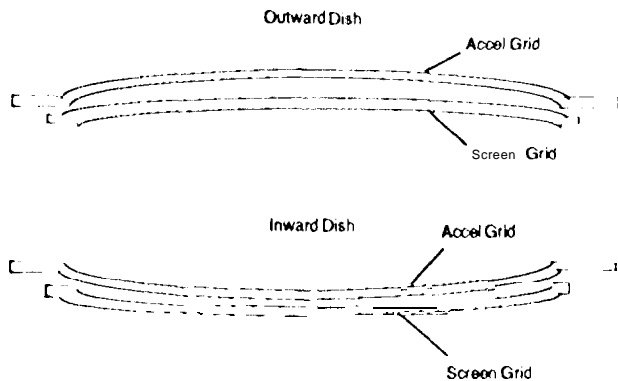


Fig. 11: Effect of Different Radii of Curvature of Screen and Accelerator grid on Grid Mounting

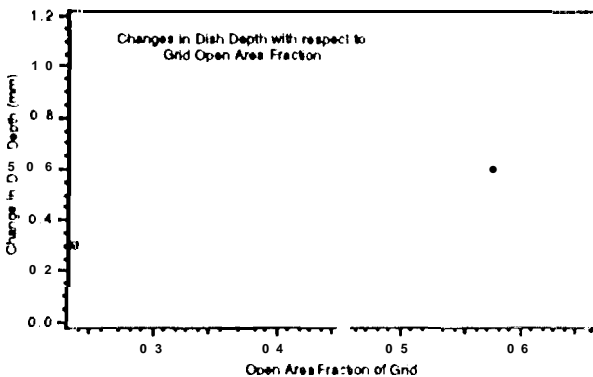


Fig. 12: Change in Dish Depth vs. Open Area Fraction

Inspection of Grid Gaps

Grid gap measurements were performed on the assembled grid set consisting of the inward dished screen and accelerator grids. The same optical measurement technique used in the grid hole determination and grid contour measurements was used. Here, the grid set was placed on the flat bench described above with the screen grid facing upward and the camera was first focused onto the screen grid top surface near the edge of a grid hole. Next, by moving the fixed-focal length camera downward, it was focused onto the accelerator grid surface beneath the screen grid and the shift in z-coordinate was determined. Due to the S1 IAG optics design of the accelerator grid a sufficient amount of grid surface was visible through the screen grid aperture. After subtracting the screen grid thickness from the observed shift in the z-coordinate the grid gap was obtained.

Table 6 lists grid gap measurements for the assembled grid set shown in Fig. 11. The x- and y-coordinates are measured again from the center of the screen grid. As can be seen, close to the grid center grid gap values of about 0.5 mm are obtained which is close to typical grid separation values. Near the edges the grid gap increases as would be expected from the difference in radius of curvature for the screen and accelerator grid. The larger grid gaps near the grid edges are obviously undesirable, however, since extracted beam current densities are lower near the edges of the beam extraction area, grid performance may be compromised less than would be the case if the larger grid gaps occurred near the grid center, depending on the ion beam current density profile.

Grid Testing

Test Set-up

The grid set shown in Fig. 8 was mounted to a J-series type thruster^{13,14} that has been modified for inert gas operation as described in Ref. 15. Figure 13 shows the operating J-series thruster with the inward dished carbon-carbon grid set attached. Tests were performed in a 2.4-m diameter and 5.5-m long stainless steel vacuum chamber equipped with three oil diffusion pumps. The pumping speed on xenon is approximately 14,000 l/s. Vacuum chamber pressure measurements were made with an ion gauge directly calibrated on xenon. Typical vacuum pressures during engine operation were $2.5\text{--}4 \times 10^{-3}$ Pa ($2\text{--}3 \times 10^{-5}$ torr). The no-load pressure in the facility is less than 4×10^{-5} Pa (3×10^{-7} Torr).

Table 6: Grid Gap Measurements

| X (mm) | Y (mm) | Z (mm) |
|----------------|-----------|-----------|
| Vertical | | |
| 139.47 | -2.54 | 1.11 |
| 128.09 | -2.01 | 0.82 |
| 116.28 | -2.44 | 0.64 |
| 100.18 | -0.25 | 0.43 |
| 78.21 | 2.54 | 0.27 |
| 48.95 | 0.30 | 0.33 |
| 35.03 | 0.76 | 0.51 |
| 27.33 | -0.10 | 0.44 |
| 14.94 | -0.56 | 0.60 |
| 6.10 | 0.64 | 0.58 |
| -1.73 | -0.28 | 0.49 |
| -3.91 | 0.05 | 0.55 |
| -6.68 | 2.36 | 0.60 |
| -16.43 | -0.43 | 0.62 |
| -30.38 | -0.78 | 0.69 |
| -45.31 | -0.43 | 0.64 |
| -59.79 | -0.36 | 0.67 |
| -76.43 | -0.23 | 0.72 |
| -96.60 | -3.60 | 0.72 |
| -118.52 | -4.50 | 0.92 |
| Horizontal | | |
| -1.04 | -140.00 | 1.03 |
| 0.51 | -98.88 | 0.51 |
| 1.85 | -112.75 | 0.63 |
| -2.01 | -73.41 | 0.45 |
| -1.24 | -50.42 | 0.45 |
| -2.26 | -38.58 | 0.52 |
| -2.90 | -21.34 | 0.57 |
| -4.04 | 9.70 | 0.61 |
| -0.91 | 31.01 | 0.61 |
| 0.28 | 57.99 | 0.57 |
| -1.60 | 77.55 | 0.61 |
| -2.18 | 99.00 | 0.61 |
| -1.83 | 118.26 | 0.80 |
| -3.58 | 141.55 | 0.98 |
| Diagonal I-III | | |
| 93.22 | 104.06 | 0.80 |
| 71.09 | 72.52 | 0.52 |
| 39.95 | 37.62 | 0.43 |
| -30.56 | -19.43 | 0.63 |
| -64.97 | -46.02 | 0.73 |
| -109.42 | 87.73 | 1.07 |
| Diagonal IV-II | | |
| 79.78 | -112.17 | 0.98 |
| 38.13 | -48.74 | 0.33 |
| 18.67 | -27.74 | 0.42 |
| -18.03 | 16.08 | 0.64 |
| -45.19 | 56.21 | 0.66 |
| -68.86 | 89.66 | 0.85 |
| -80.44 | 112.47 | 1.20 |

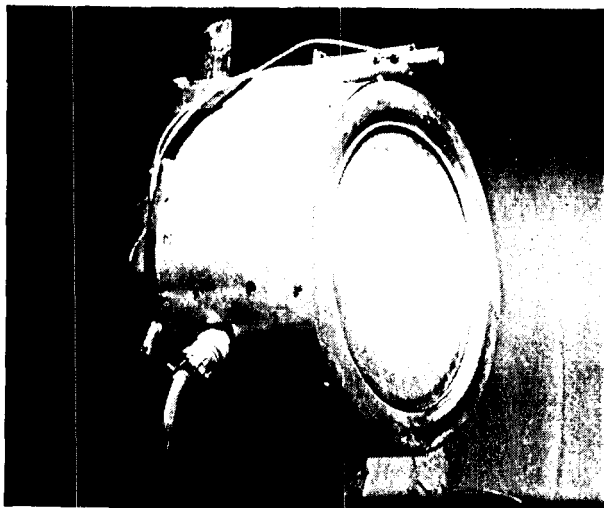


Fig. 13: 30-cm-dia. Carbon-Carbon Grid Set mounted on J-Series Type Thruster. Two-Grid System, Dished Inward.

The propellant flow control system was described in detail in Ref. 12. This flow system was designed for the segmented ion thruster (SIT) described in that reference and consisted of four parallel branches for the four segments of the SIT engine. Here, only one of the four branches was used. The same set of power supplies as used in the SIT thruster tests was used, consisting of laboratory power supplies throughout¹². For the main discharge 0-50 V and 0-20 A supplies from Power Ten, inc. are used. The high voltage supply is a custom 12 kW (6 A and 2 kV) supply made by Spellman. The cathode and neutralizer start supplies are made by Kepco.

Test Procedures

Perveance Measurements

The perveance of an ion engine grid describes its ability to extract current from the discharge chamber. Perveance measurements were performed by keeping the screen current (beam current plus accelerator impingement current) constant while varying the total grid voltage between the screen and accelerator grids and measuring the accelerator impingement current. In order to ensure a constant screen current, the discharge current was adjusted accordingly. The screen current is slightly larger than the beam current by an amount equal to the accelerator impingement current.

Electron Backstreaming Measurements

Electron backstreaming occurs when the (negative.) accelerator grid voltage becomes too small (in

magnitude) to shield the discharge chamber plasma from electrons in the beam plasma. Backstreaming electrons cause an increase in screen grid current, indicating that the electron backstreaming limit has been reached. Here, accelerator voltage was lowered manually to the point of electron backstreaming onset.

Results

Perveance Measurements

Figure 14 shows the perveance data obtained with the inward dished 30-cm carbon-carbon grid set at screen currents of 1.7 A and 1.0 A. In both cases screen current varied slightly during the course of taking the data shown in Fig. 4. Perveance limits for the 1.0 A screen current are around 1 kV and for the 1.7 A case around 1.4 kV. These values are compared in Fig. 14 with values obtained previously^{1,2} for a molybdenum grid set mounted to the same thruster. The molybdenum grid set consisted of a screen and accelerator grid with same hole geometries as the grids used in the carbon-carbon set. However, the molybdenum set was dished outward and a smaller grid gap of 0.66 mm throughout the system. As a result, perveance limits were lower for the molybdenum set. At 1.76 A and 1.01 A beam currents the perveance limits were 1.15 kV and about 980 V. Due to the shallow knee in the 1.01 A curve for the molybdenum set it is difficult to determine the perveance limit precisely. The higher accelerator grid currents for the carbon-carbon grid set are a facility effect. The carbon-carbon test was performed at a background pressure of $2-4 \times 10^{-3}$ Pa ($2-3 \times 10^{-5}$ Torr) while the molybdenum test was conducted in a different, cryo-pumped facility at a back pressure of merely 5×10^{-4} Pa (4×10^{-6} torr), resulting in lower charge exchange ion production.

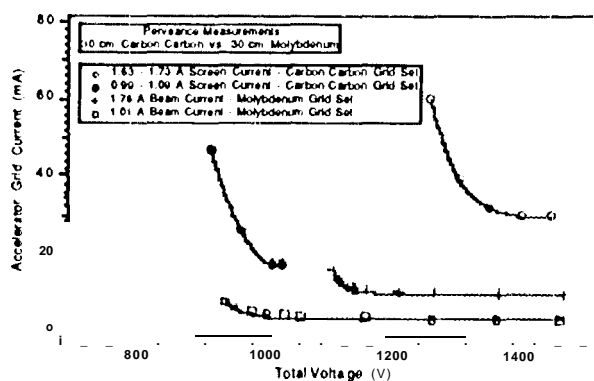


Fig. 14: Perveance Measurements for 30-cm dia. Dished Carbon-Carbon and Molybdenum Grids

Electron Backstreaming Measurements

Electron backstreaming data are shown in Fig. 15. The minimum accelerator voltage at a screen current of about 1 A is -90 V. The beam voltage was 950 V in this case, translating into a maximum net-to-total voltage ratio of 0.91. For 1.7 A screen current, the minimum accelerator voltage was found to be around -130 V. Here the beam voltage was 1250 V, corresponding to a maximum net-to-total voltage ratio of 0.906, i.e. slightly less than for the 1 A value. The corresponding values for the closer gapped molybdenum set mentioned above were -144 V and -121 V at beam currents of 1.76 A and 1.01 A, respectively.

Conclusions

Two 30-cm carbon-carbon grid sets were fabricated, including two light-weight carbon-carbon mount rings, in order to study the feasibility of fabrication, assembly and operation of large diameter carbon-carbon grid. Screen, accelerator and decelerator grid optics were manufactured in order to also test the carbon-carbon grids in SAND optics configuration. The grids were designed to be retrofitable to the current 30-cm NSTAR engine design.

The carbon-carbon panels were fabricated from unidirectional tape using a $[0^\circ/460^\circ/-60^\circ]$ lay-up resulting in the strongest quasi-isotropic carbon-carbon known to the authors. High fiber volume fractions of 57% led to a flexural modulus for these panels of up to 275 MPa or about 80% of the corresponding value for molybdenum. High flexural moduli are a crucial material property for

ion engine grid materials since stronger grids will flex to a lesser degree due to electrostatic stresses or launch vibrations.

A laser-machining process was used to machine grid hole apertures. Grid contour measurements performed after fabrication revealed that radii of curvature obtained for the grids were larger than specified, with larger radii of curvature correlated with higher grid open area fractions. Screen, accelerator and decelerator radii of curvature were 23.7 cm, 21.4 cm and 22.2 cm, respectively, compared to the specified design value of 20.3 cm. Actual radii of curvature were estimated based on height measurements at the grid center, assuming a spherical grid contour. However, small differences in grid height at opposite locations of the grid were noted and are on the order of 0.1 mm for the accelerator grid and 0.3 - 0.4 mm for the decelerator grid could be noted, indicative of a slight warping of the grids.

The large difference in radii of curvature between the screen and accelerator grid dictated that the grids be mounted dishd inwards in order to obtain the desired grid-to-grid separation on the centerline. Perveance and electron backstreaming were performed with this grid set. The perveance limit at a 1 A screen current is approximately 1 kV and for a 1.7 A screen current it was around 1.4 kV.

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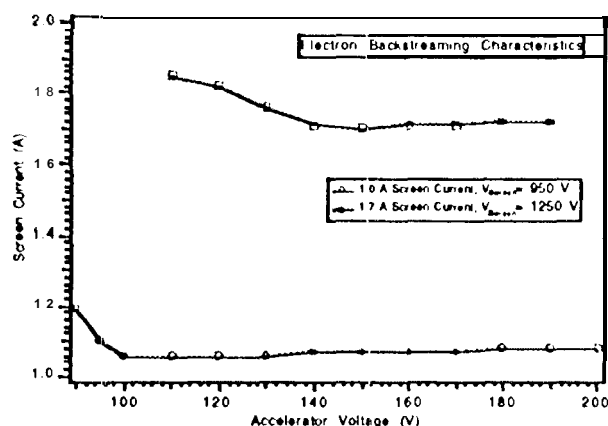


Fig. 15: Electron Backstreaming at Screen Currents of 1.7 A and 1.0A.

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